### Thin and Thick Pressure Vessels

Unit 4

- <u>Classify</u> a pressure vessel as thin-walled or thick-walled
- <u>Calculate</u> the stress (hoop stress) in a sphere subject to an internal pressure
- <u>Calculate</u> the stress (hoop stress and longitudinal stress) in a cylinder subject to an internal pressure
- <u>Determine</u> the required wall thickness of a pressure vessel to safely resist a given internal pressure

# **Chapter Objectives**

#### • Terminology:

- R<sub>i</sub>, R<sub>o</sub>, R<sub>m</sub>, Inside, outside and mean (average) radii
- $D_i$ ,  $D_o$ ,  $D_m$ , Inside, outside and mean (average) diameters
- t wall thickness
- If R<sub>m</sub> / t ≥ 10, pressure vessel is considered <u>thin-walled</u>
  In terms of the diameter; D<sub>m</sub> / t ≥ 20
- Derivation of formulas for stresses in thin-walled pressure vessels are based on the <u>assumption</u> that the stresses are <u>constant</u> throughout the wall of the vessel
- If R<sub>m</sub> / t ≥ 10, pressure vessel is considered <u>thick-walled</u>
  Stresses in thick-walled pressure vessels are <u>not</u> constant throughout the wall of the vessel

#### **Thin-walled Pressure Vessels**

- Internal pressure in sphere acts perpendicular to the surface
  - Uniform over the interior surface
- Cut Free-Body Diagram through center of sphere
- Internal forces in walls appear on FBD
  - Since FBD was cut through center of sphere, these forces are horizontal
- For vertical equilibrium:  $\Sigma F_Y = 0$ 
  - Vertical components of internal pressure are equal and opposite
  - Vertical components in opposite directions cancel each other
- For horizontal equilibrium:  $\Sigma F_X = 0$ 
  - Internal force in wall must equal the resultant horizontal force due to internal pressure

# **Thin-walled Spheres**

#### Considering horizontal components of internal pressure

- Resultant force  $F_R = p A_P$ 
  - $A_p = projected$  area of sphere on plane cut through the diameter =  $\pi D_m^2 / 4$
- Since  $\Sigma F_X = 0$ , internal force in wall =  $F_R$
- Stress in wall:  $\sigma = F/A = F_R / A_W$ 
  - A<sub>W</sub> = Area of sphere wall

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$$A_W = \pi D_o^2 / 4 - \pi D_i^2 / 4 = \pi (D_o^2 - D_i^2) / 4$$

- For a thin-walled sphere
  - $A_W \approx \pi D_m t$  --> the area of a strip of thickness = t and length = average circumference ( $\pi D_m$ )
- Stress in wall of sphere
  - $\sigma = F_R / A_W = p A_P / A_W = p (\pi D_m^2 / 4) / \pi D_m t = p D_m / 4 t$

#### **Thin-walled Spheres**

- Cylinders used as pressure vessels and for piping of fluids under pressure
- Two types of stresses
  - Longitudinal stress along the long axis of the cylinder
  - Hoop stress (tangential stress) around the circumference of the cylinder

## **Thin-walled Cylinders**

#### Longitudinal Stress

- Cut Free-Body Diagram through cylinder, perpendicular to longitudinal axis
- Longitudinal internal forces in walls appear on FBD
  - Forces are horizontal
- For horizontal equilibrium: ∑ F<sub>X</sub> = 0
  Internal force in wall must equal the resultant horizontal force due to internal pressure

# **Longitudinal Stress in Thin-walled Cylinders**

- If end of cylinder is closed, resultant force  $F_R = p A = p \pi D_m^2 / 4$ 
  - Since  $\Sigma F_X = 0$ , longitudinal internal force in wall =  $F_R$
- Stress in wall:  $\sigma = F/A = F_R / A_W$ 
  - A<sub>W</sub> = Area of sphere wall
  - $A_W = \pi D_o^2 / 4 \pi D_i^2 / 4 = \pi (D_o^2 D_i^2) / 4$
- For a thin-walled cylinder
  - $\underline{A}_W \approx \pi D_m t$ ,
  - The area of a strip of thickness = t and length = average circumference ( $\pi D_m$ )
- Longitudinal stress in wall of cylinder
  - $\sigma = F_R / A_W = p A_P / A_W = p (\pi \hat{D}_m^2 / 4) / \pi D_m t = p D_m / 4$

Longitudinal stress is <u>same</u> as stress in a sphere

# **Longitudinal Stress in Thin-walled Cylinders**

#### Isolate a ring of length L from the cylinder

- Cut a vertical section through ring, passing through its center
- Draw a FBD of segment either side of section
- Similar to analysis of sphere, resultant force  $F_R = p A_P$ 
  - $A_P = projected$  area of ring =  $D_m L$
- Stress in wall:  $\sigma = F/A = F_R / A_W$ 
  - $A_W$  = Cross-sectional area of cylinder wall = 2 t L
- Hoop stress in wall of cylinder
  - $\sigma = F_R / A_W = p A_P / A_W = p D_m L / 2 t L = p D_m / 2 t$
  - Hoop stress is twice the magnitude of longitudinal stress
  - Hoop stress in the cylinder is also twice the stress in a sphere of the same diameter carrying the same pressure

### Hoop Stress in Thin-walled Cylinders